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FORTTRAN programs to calculate differential stress concentration factors
around two-dimensional circular and elliptical inclusions
in an edge-loaded plate

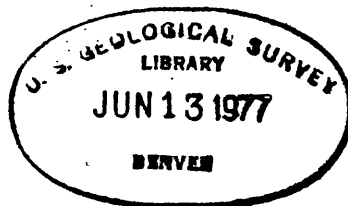
by

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Abstract

FORTTRAN IV listings and check-case outputs are given for user-interactive programs CIRCLE and ELLIPSE. The programs suppose either circular or elliptical inclusions, respectively, in an edge-loaded elastic plate, and calculate differential stress concentration factors over a 10 x 11 array of physical points which can be specified by the user. Differential stress concentration factor is defined in this case to be the amount by which the radius of Mohr's circle expands in the vicinity of the inclusion. The two programs handle two-dimensional geometry only. Both programs print extensive prompt-lists to help the user on input.

Equations for stresses in the vicinity of a circular cylindrical elastic inclusion in an edge-loaded elastic plate may be found in Jaeger and Cook¹. Similarly, the solution for an elliptical, cylindrical inclusion is given by Donnell². In both cases, the generators of the cylinder are taken perpendicular to the plane of the stresses, so that the problems under consideration are strictly two-dimensional. The stresses depend on the contrast in rigidity, $\frac{E_I}{E_P}$, between inclusion and plate, and on the Poisson's ratios, ν_P and ν_I , of plate and inclusion, respectively.

In problems of failure mechanics, it is of interest to know (a) the direction of stress trajectories and (b) the amount by which the

radius of Mohr's circle has increased due to the inclusion. Programs CIRCLE and ELLIPSE were written using the equations given by Jaeger and Cook and by Donnell, respectively, to calculate these two quantities. In addition, the Donnell equations were generalized somewhat to allow plate and inclusion to have differing Poisson's ratios, and to allow calculation of stresses at all points when the applied stress is at right angles to the ellipse's semi-major axis.

It is assumed that principal stresses P_1 and P_3 , where $P_1 > P_3$, are applied on the plate's edge. Tensional stresses are positive. Thus, a uniaxial compression of one unit would have $P_1 = 0$, $P_3 = -1$, and the radius of Mohr's circle everywhere in the plate if the inclusion were absent would be $R = (P_1 - P_3)/2 = 0.5$. We define the "differential stress concentration factor" at a given point as the radius of Mohr's circle in the presence of the inclusion divided by its radius in its absence, or

$$F = \pm \sqrt{[(\sigma_{\alpha\alpha} - \sigma_{\beta\beta})/2]^2 + \tau_{\alpha\beta}^2/2}$$

where $\sigma_{\alpha\alpha}$ and $\sigma_{\beta\beta}$ are calculated normal stresses and $\tau_{\alpha\beta}$ the calculated shear stress of a unit prism at the point. As a flag to the user, the sign of F is output as negative in case $\sigma_{\alpha\alpha} + \sigma_{\beta\beta}$ has opposite sign from $P_1 + P_3$. A switch is provided to normalize factor F by $(P_1 + P_3)/2$ rather than by R when the user wishes--for example, to calculate the case $P_1 = P_3$; however, then F 's simple meaning as Mohr circle amplification factor is lost.

In program CIRCLE, principal stresses P_1 and P_3 are renamed P_X and P_Y , and are associated with x and y -axes. CIRCLE allows either P_X or P_Y to be the larger. Differential stress concentration factors are

calculated over a 10 x 11 grid of circular cylindrical coordinate points (RPERA, THETA) which may be set by the user, but have default values in one quadrant which are probably adequate for most investigations. (Values repeat by symmetry through the other quadrants.)

In program ELLIPSE, the convention $P1 > P3$ should be adhered to. Results are again calculated over an adjustable 10 x 11 grid, whose default values cover one quadrant. Elliptical coordinates (ALPHA, THETA) are used, where

$$x = C \cosh (\text{ALPHA}) \cos (\text{THETA}),$$

$$y = C \sinh (\text{ALPHA}) \sin (\text{THETA}),$$

$$C = \sqrt{1 - E^2},$$

and E is the ellipticity (= semi-minor axis/semi-major axis) of the ellipse. The angle THETA is in degrees. The x-axis is along the long, or semi-major, axis of the ellipse. Default values of ALPHA step in 10 equal increments from the inclusion boundary to a value whose x-intercept corresponds to 3 times the semi-major axis. Input variable ANGLE is the clockwise angle in degrees between the x-axis and the direction of $P1$. Figure 1 illustrates the resulting geometrical relations for the case $\text{ALPHA} = \pm 30$, when the plate is under uniaxial unit compression. Unless $\text{ANGLE} = 0^\circ$ or 90° , a second computation with the sign of ANGLE reversed must be run in order to see the complete stress concentration pattern, since the results of a given run repeat by symmetry in these cases to only one other quadrant (the opposite one), and not to all three.

The programs were written in FORTRAN IV and were run on the U.S. Geological Survey Honeywell 68/80 Multics computer at Denver, Colorado. They were written for user-interactive use, and print out many messages to prompt appropriate user replies. Input is by FORTRAN namelist. Listings

of the programs and some check-cases follow. Two non-standard features of Multics FORTRAN seen on the listings are continuation lines without a continuation character, and the use of both upper and lower case letters.

References cited

1. Jaeger, J. C., and Cook, N. G. W., 1969, Fundamentals of Rock Mechanics: Methuen, p. 236-251.
2. Donnell, L. H., 1941, Stress concentrations due to elliptical discontinuities in plates under edge forces: Theodore von Karman Anniversary Volume, Pasadena, p. 293-309.

PROGRAM CIRCLE

```

calculation of stress concentration around a circular inclusion in a plate.....
**
c.....equations from Jaeger and Cook, Fundamentals of Rock Mechanics,
c      (Methuen, 1969), Section 10.9, pages 243-251.
      real norm
      integer problm,switch,detail
      automatic rpera,theta
      dimension bigf(11,10),rpera(11),theta(10),traj(11,10),sigmar(11,10),sigmat
(11,10),taurt(11,10)
      namelist /input/xk,prp,pri,px,py,problm,switch,detail
      namelist /grid/theta,rpera
c.....Set default input values.
      problm=1
      switch=1
      px=-1.
      py=0.
      deg2rq=3.14159/180.
      data (rpera(i),i=1,11)/1.0,1.1,1.2,1.3,1.4,1.5,1.6,1.8,2.0,2.5,3.0/
      data (theta(i),i=1,10)/0.,10.,20.,30.,40.,50.,60.,70.,80.,90./
c.....Print user prompt messages.
      -print 75
75      format (////" Stress Concentration around a Circular Inclusion."
// " Program by Dave Campbell, US Geol. Survey, Denver. Feb77"
// " Equations from Jaeger and Cook, Section 10.9,"
// " Variables in namelist input:"
// "      xk = (Rigidity of inclusion)/(Rigidity of plate)"
// "      prp = Poisson's ratio of plate."
// "      pri = Poisson's ratio of inclusion."
// "      px = Force in x-direction on plate's y-edges. (default=-1.)"
// "      py = Force in y-direction on plate's x-edges. (default=0.)"
// "      (Note: negative forces are compressional, positive ones tensional.)")
      print 76
76      format (
// "      - problm = 1 for plane strain (default),"
// "                2 for plane stress,"
// "                0 to stop."
// "      detail = 0 to output stress concentration factor only. (default)"
// "                1 to output stress trajectory angles, too."
// "                2 to output stress fields, too."
// "      switch = 1 to normalize by (px-py)/2 (default)"
// "                2 to normalize by (px+py)/2.")
c.....Subsequent passes enter here.
400      continue
      print 10
10      format(////" type namelist input:xk,prp,pri...(problm=0 to stop.)",/)
      read input
      if(problm.eq.0)stop
      t=(px+py)/2.
      s=(px-py)/2.
      norm=s
      if (switch .eq. 2) norm=t
      yo=3.-4.*prp
      yi=3.-4.*pri
      if (problm .eq. 1) go to 100
      yo=(3.-prp)/(1.+prp)
      yi=(3.-pri)/(1.+pri)
100      continue
      d1=xk*yo+1.
      d2=2.*xk*yi-1.
      a=2.*(1.-xk)/d1

```

```

b=(yi-1.-xk*(yo-1.))/d2
c=(xk-1.)/d1
f=2.*d1*d2
e=xk*(yo+1.)
h1=((xk*(yo+2.)+yi )*e)/f
h2=((xk*(yo-2.)-yi+2.)*e)/f
c.....Y & C eqn (16).
sigma1=h1*px+h2*py
c.....Y & C eqn (17).
sigma2=h2*px+h1*py
y=(sigma1-sigma2)/2.
fbig=(sign(y,sigma1+sigma2))/norm
do 1000 i=1,11
xi=rpera(i)
x=1./(xi**2)
sr=t*(1.-b*x)
st=t*(1.+b*x)
tc=3.*c*x*x
fr=s*(1.-2.*a*x-tc)
ft=s*(1.-tc)
tt=s*(1.+a*x+tc)
do 1000 j=1,10
tj=theta(j)
ttt=2.*tj*deg2rd
cst=cos(ttt)
snt=sin(ttt)
c.....Y & C eqn (20).
sigr=sr+fr*cst
c.....Y & C eqn (21).
sigt=st-ft*cst
c.....Y & C eqn (22).
tau=-tt*snt
sigmar(i,j)=sigr
sigmat(i,j)=sigt
taurt(i,j)=tau
y=sqrt(((sigr-sigt)/2. )**2+tau**2)
bigf(i,j)=(sign(y,sigr+sigt))/(sign(norm,t))
traj(i,j)=atan2(2.*tau,sigr-sigt)/(2.*deg2rd)+theta(j)
1000 continue
c.....printing of output begins here.
print 40
40 format(////" DIFFERENTIAL STRESS CONCENTRATION FACTORS AROUND A CIRCULAR I
NCLUSION",/)
if (problem .eq. 1) print 62
if (problem .eq. 2) print 64
62 format(1h ,29x," plane strain calculation")
64 format(1h ,29x," plane stress calculation")
print 45, xk,prp,pri
45 format(1h ,29x,"inclusion rigidity/plate rigidity = ",f8.3,/30x,"Poisson's
ratio of plate = ",f8.3,/30x,"Poisson's ratio of inclusion = ",f8.3)
if (switch .eq. 0) print 46
46 format (30x,"normalization factor=(px+py)/2")
print 30,(theta(k),x=1,10)
30 format("/" theta=",10f7.2)
print 35
35 format(" r/a !",72(1h_))
print 25, fbig
25 format (1h ,<1.0 !",f7.3," everywhere.....")
print 20, (rpera(i),(bigf(i,j),j=1,10),i=1,11)
20 format(1h ,f4.1,x,1h!,10f7.3)

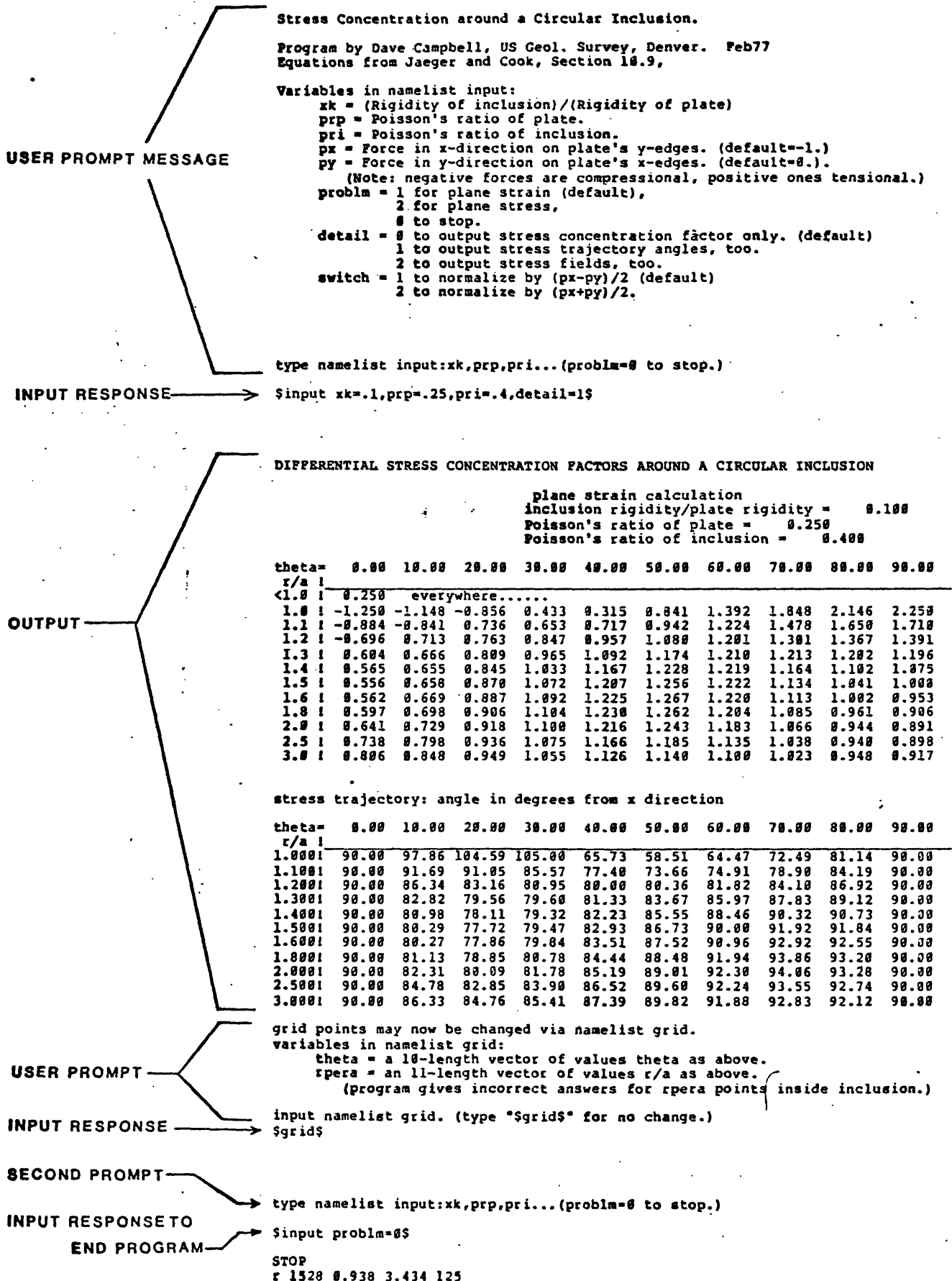
```

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        if (detail .eq. 0) go to 300
        print 39
39      format (//" stress trajectory: angle in degrees from x direction")
        print 30,(theta(k),k=1,10)
        print 35
        print 21, (rpera(i),(traj(i,j),j=1,10),i=1,11)
21      format(1h,f5.3,1h!,10f7.2)
        if (detail .eq. 1) go to 300
        print 36
36      format ( //" rpera component of stress:")
        print 30,(theta(k),k=1,10)
        print 35
        print 20, (rpera(i),(sigmar(i,j),j=1,10),i=1,11)
        print 37
37      format(//" theta component of stress:")
        print 30,(theta(k),k=1,10)
        print 35
        print 20, (rpera(i),(sigmat(i,j),j=1,10),i=1,11)
        print 38
38      format (//" tau component of stress:")
        print 30,(theta(k),k=1,10)
        print 35
        print 20, (rpera(i),(taurt(i,j),j=1,10),i=1,11)
c.....Now allow user to change gridpoints, in order to investigate details
c      in critical regions.
300     continue
        print 80
80      format (/" grid points may now be changed via namelist grid."
/" variables in namelist grid:"
/"      theta = a 10-length vector of values theta as above."
/"      rpera = an 11-length vector of values r/a as above."
/"      (program gives incorrect answers for rpera points inside inclusion.)"
/" input namelist grid. (type ",8h"$grid$"," for no change.)")
        read grid
        go to 400
        end

```


EXAMPLE AND CHECK CASE: CIRCLE



PROGRAM ELLIPSE

calculation of stress concentration around an elliptical inclusion.....

c .. equations from donnell, 1941, von karman volume, pps. 293-309.

real norm

integer problm, switch, detail

automatic alpha, theta

dimension bigf(11,10), traj(11,10), alpha(11), theta(10), signaa(11,10), signab(11,10), tauab(11,10)

common /geom/xxk,yo,yi,a0

common /point/e2a,em2a,e4a,en4a,c2b,s2b,c4b,s4b

namelist/input/xxk,prp,pri,p1,p3,angle,e,problm,switch,detail

namelist/grid/theta,alpha

acosh(x)=alog(x+sqrt(x**2-1.))

c.....set default input values.

data (theta(i),i=1,10)/0.,10.,20.,30.,40.,50.,60.,70.,80.,90./

problm=1

switch=1

detail=0

p1=0.

p3=-1.

— angle=90.

e=.5

deg2rd=3.14159/180.

itime=0

c.....print user prompt messages.

print 75

75 format (////" Stress Concentration around Elliptical Inclusion."

/" Program by Dave Campbell, US Geol. Survey, Denver. Feb77"

/" Equations from donnell, 1941, von karman volume, pps.293-309."

/" Variables in namelist input:"

/" xk = (Rigidity of inclusion)/(Rigidity of plate)"

/" pri = Poisson's ratio of inclusion."

/" prp = Poisson's ratio of plate."

/" p1 = maximum principal stress on plate at infinity. (default=0.)"

/" p3 = minimum principal stress on plate at infinity. (default=-1.)"

/" angle = angle in degrees between p1 and ellipse major axis. (default=90

)"

/" (negative stresses are compressional, positive ones tensional."

/" hence default case has compression along ellipse's major axis.)"

print 76

76 format(

" e = ellipticity of ellipse (=semi-minor axis/semi-major axis)."

/" (Must be between 0.(thin sawcut) and 1.(circle). default=.5)"

/" problm = 1 for plane strain (default),"

/" 2 for plane stress,"

/" 0 to stop."

/" detail = 0 to output stress concentration factor only. (default)"

/" 1 to output stress trajectory, angles, too."

/" 2 to output stress fields, too."

/" switch = 1 to normalize by mohr circle radius (default)"

/" 2 to normalize by hydrostatic stress (p1+p3)/2.)"

c.....subsequent passes enter here.

400 continue

print 10

10 format(////" type namelist input:xxk,pri,prp...(problm=0 to stop.)",/)

read input

if(problm.eq.0)stop

c.....Break down general case of ellipse at angle to principal stress

c directions in terms of bndry loads px,py, 3 tf, for which

c Donnell gives solutions.

convention: x-axis is taken along ellipse's major axis.

```

c      px = Normal Force in x-direction on plate's y-edges.
c      py = Normal force in y-direction on plate's x-edges.
c      (Note: negative forces are compressional, positive ones tensional.)
c      tf = Tangential (shear) force on plate's edges.
c      (Note: equal and opposite on adjacent edges so plate won't rotate.)
      ang=angle*deg2rd*2.
      t=(p1+p3)/2.
      s=abs((p1-p3)/2.)
      px=t+s*cos(ang)
      py=t-s*cos(ang)
      tf=t*sin(ang)
      norm=sign(s,t)
      if (switch .eq. 2) norm=t
c.....Donnell takes prp=pri. The next 7 steps, plus changes noted
c      in subroutines pcons & tcons, set up the general case.
      xxk=xk*(1.+prp)/(1.+pri)
      yo=3.-4.*prp
      yi=3.-4.*pri
      if (problem .eq. 1) go to 100
      yo=(3.-prp)/(1.+prp)
      yi=(3.-pri)/(1.+pri)
100  continue
c.....Set up default values of alpha, running from bndry to x-
c      value 3 times semi-major axis. Skip on subsequent passes.
      if (itime .eq. 1) go to 200
      a0=acosh(1./sqrt(1.-e**2))
      alpha(1)=a0
      alpha(11)=acosh(3./sqrt(1.-e**2))
      ainc=(alpha(11)-a0)/10.
      do 1002 i=2,10
1002  alpha(i)=alpha(i-1) + ainc
200  continue
      call pt(a0,0.)
      call pcons(+1,py,y1,y1py2,y2py3,y3py4,y4py5,y5p)
      call tcons(tf,t1,t1p,t2,t2p,t3,t3p,t4,t4p,t5,t5p)
      call pcons(-1,px,x1,x1px2,x2px3,x3px4,x4px5,x5p)
      call pt(a0/2.,0.)
      call pterm(y1py2py3py4py5p,ysiga,ysigb,ytau)
      call tterm(t1p,t2p,t3p,t4p,t5p,tsiga,tsigb,ttau)
      call pterm(x1px2px3px4px5p,xsiga,xsigb,xtau)
      siga=xsiga+ysiga+tsiga
      tau=xtau+ytau+ttau
      sigb=xsigb+ysigb+tsigb
      s=siga+sigb
      y=sqrt(((siga-sigb)/2.)**2 + tau**2)
      fbig=sign(y,s)/norm
      do 1000 j=1,10
      tj=theta(j)
      beta=tj*deg2rd
      do 1000 i=1,11
      alfa=alpha(i)
      call pt(alfa,beta)
      call pterm(y1,y2,y3,y4,y5,ysiga,ysigb,ytau)
      call tterm(t1,t2,t3,t4,t5,tsiga,tsigb,ttau)
      call pterm(x1,x2,x3,x4,x5,xsiga,xsigb,xtau)
      if (alfa .ge. a0) go to 500
      call pterm(y1py2py3py4py5p,ysiga,ysigb,ytau)
      call tterm(t1p,t2p,t3p,t4p,t5p,tsiga,tsigb,ttau)
      call pterm(x1px2px3px4px5p,xsiga,xsigb,xtau)

```

```

500  continue
c.....Do superposition of px,py, & tf solutions.
      siga=xsiga+ysiga+tsiga
      tau=xtau+yttau+tttau
      sigb=xsigb+ysigb+tsigb
      sigmaa(i,j)=siga
      sigmab(i,j)=sigb
      tauab(i,j)=tau
      s=siga+sigb
      y=sqrt(((siga-sigb)/2.)*2 + tau**2)
      bigf(i,j)=sign(y,s)/norm
      traj(i,j)=(atan2(2.*tau,siga-sigb)/2.
      +atan2((exp(alfa)+exp(-alfa))*sin(beta),(exp(alfa)-exp(-alfa))*cos(beta))
      )/deg2rd
1000  continue
c.....output section begins here.
      print 40
40    format(////" DIFFERENTIAL STRESS CONCENTRATION FACTORS AROUND AN ELLIPTICA
L INCLUSION",/)
      if (problem .eq. 1) print 62
      if (problem .eq. 2) print 64
62    format(1h ,29x,"plane strain calculation")
64    format(1h ,29x,"plane stress calculation")
      print 45, xk,prp,prrie,p1,p3,angle
45    format(1h ,29x,"inclusion rigidity/plate rigidity = ",f8.3,/30x,"Poisson's
ratio of plate = ",f8.3,/30x,"Poisson's ratio of inclusion = ",f8.3,/30x,"ellip
ticity of ellipse = ",f8.3,/30x,"principal stress p1 = ",f8.3,/30x,"principal st
ress p3 = ",f8.3,/30x,"ellipse tilt angle = ",f4.0," deg")
      if (switch .eq. 0) print 46
46    format(30x,"normalization factor=(p1+p3)/2")
      print 30,(theta(k),k=1,10)
30    format(/" theta=",10f7.2)
      print 35
35    format(" alpha!",72(1h!))
      print 25,fbig
25    format(" <a0 !",f7.3," everywhere.....")
      print 20, (alpha(i),(bigf(i,j),j=1,10),i=1,11)
20    format(1h ,f5.3,1h!,10f7.3)
      itime=1
      if (detail .eq. 0) go to 300
      print 39
39    format (// " stress trajectory: angle in degrees from x direction (along ma
jor axis)")
      print 30,(theta(k),k=1,10)
      print 35
      print 21, (alpha(i),(traj(i,j),j=1,10),i=1,11)
21    format(1h ,f5.3,1h!,10f7.2)
      if (detail .eq. 1) go to 300
      print 36
36    format ( // " alpha component of stress:")
      print 30,(theta(k),k=1,10)
      print 35
      print 20, (alpha(i),(sigmaa(i,j),j=1,10),i=1,11)
      print 37
37    format(// " beta component of stress:")
      print 30,(theta(k),k=1,10)
      print 35
      print 20, (alpha(i),(sigmab(i,j),j=1,10),i=1,11)
      print 38
38    format (// " tau component of stress:")

```

```

      print 30,(theta(k),k=1,10)
      print 35
      print 20, (alpha(i),(tauab(i,j),j=1,10),i=1,11)
c.....Now allow user to change grippoints so he can look at details
c      of the solution in small regions.
300  continue
      print 80
80   format (/" grid points may now be changed via namelist grid."
/" variables in namelist grid:"
/"      theta = a 10-length vector of values theta as above."
/"      alpha = an 11-length vector of values alpha as above."
/" input namelist grid. (type ",8h"$grid$"," for no change.)")
      read grid
      go to 400
end

```

```

subroutine pt(a,b)
common/point/e2a,em2a,e4a,em4a,c2b,s2b,c4b,s4b
e2a=exp(2.*a)
em2a=exp(-2.*a)
e4a=exp(4.*a)
em4a=exp(-4.*a)
c2b=cos(2.*b)
s2b=sin(2.*b)
c4b=cos(4.*b)
s4b=sin(4.*b)
return
end

```

```

subroutine pterm(ap1,am1,b0,bp2,bm2,siga,sigb,tau)
c.....this subroutine programs donnell's eqns (14).
common /point/e2a,em2a,e4a,em4a,c2b,s2b,c4b,s4b
factr=1./(((e2a+em2a)/2.)-c2b)**2
siga=factr*(-ap1*((2.+e4a)-2.*(e2a+em2a)*c2b+c4b)
+am1*((2.+em4a)-2.*(e2a+em2a)*c2b+c4b) - b0*(e2a-em2a)
-bp2*(3.*e2a-(3.+e4a)*c2b+em2a*c4b)
+bm2*(3.*em2a-(3.+em4a)*c2b+em2a*c4b))
sigb=factr*(-ap1*((2.+e4a)-4.*e2a*c2b+c4b)
+am1*((2.+em4a)-4.*em2a*c2b+c4b) + b0*(e2a-em2a)
+bp2*(3.*e2a-(3.+e4a)*c2b+em2a*c4b)
-bm2*(3.*em2a-(3.+em4a)*c2b+em2a*c4b))
tau=-factr*((ap1+am1)*(e2a+em2a)+2.*b0)*s2b
+bp2*((3.+e4a)*s2b-em2a*s4b)+bm2*((3.+em4a)*s2b-em2a*s4b))
return
end

```

```

subroutine tterm(ap1,am1,b0,bp2,bm2,siga,sigb,tau)
c.....this subroutine programs donnell's eqns (21).
common /point/e2a,em2a,e4a,em4a,c2b,s2b,c4b,s4b
factr=1./(((e2a+em2a)/2.) - c2b)**2
siga=factr*(-(ap1+am1)*(2.*(e2a+em2a)*s2b-s4b)-2.*b0*s2b
-bp2*((3.+e4a)*s2b-em2a*s4b) - bm2*((3.+em4a)*s2b-em2a*s4b))
sigb=factr*((ap1+am1)*s4b+2.*b0*s2b+bp2*((3.+e4a)*s2b-em2a*s4b)
+bm2*((3.+em4a)*s2b-em2a*s4b))
tau=factr*((ap1+am1)*c2b+b0)*(e2a-em2a)+bp2*(3.*e2a-(3.+e4a)*c2b
+em2a*c4b)-bm2*(3.*em2a-(3.+em4a)*c2b+em2a*c4b))
return
end

```

```

      subroutine pcons(n,py,ap1,ap1pr,am1,am1pr,b0,b0pr,bp2,bp2pr,bm2,bm2pr)
c.....This subroutine programs Donnell's eqns (15), generalized to in-
clude additional factor n. His eqns hold only if load is in y dire-
ction, and n=1. When load is in x direction, signs change, and n=-1.
c.....A second generalization, to include unmatched Poisson's ratios,
changes y0 to yi in the denominator of the ap1pr expression.
      common /geom/xk,yo,yi,a0
      common /point/e2a,em2a,e4a,em4a,c2b,s2b,c4b,s4b
      ap1=-py/5.
      bp2=ap1*n
      if (xk .eq. 0) go to 100
      bigm=(yi-1.-xk*(yo-1.))/(xk*(yo+1.))
      ap1pr=((xk*yo+1.)*e2a+(1.-xk)*(2.*n+em2a))*ap1
      /((yi+1.)*em2a+(xk*yo+1.)*(bigm+1.)*(e2a-em2a))
      am1=-e4a*ap1-(bigm+1.)*(1.-e4a)*ap1pr
      b0=-bigm*(e2a-em2a)*ap1pr
200  continue
      am1pr=-ap1pr
      bp2pr=n*ap1+em2a*(ap1-am1-2.*ap1pr)/2.
      bm2pr=-bp2pr
      b0pr=0.
      bm2=-n*e4a*ap1-(1.-e4a)*bp2pr
      return
100  continue
      ap1pr=0.
      am1=-ap1*(em2a+(1.-e4a)*(e2a+em2a+2.)/(e2a-em2a))
      b0=-ap1*(e2a+em2a+2.)
      go to 200
      end

```

```

      subroutine tcons(tf,ap1,ap1pr,am1,am1pr,b0,b0pr,bp2,bp2pr,bm2,bm2pr)
c.....This subroutine programs Donnell's eqns (24), generalized to in-
clude cases when Poisson's ratios of plate and inclusion don't match.
c.....Only eqns for am1pr & ap1pr are effected by this generalization.
      common /geom/xk,yo,yi,a0
      common /point/e2a,em2a,e4a,em4a,c2b,s2b,c4b,s4b
      f=(1.+yo)/(1.+yi)
      ap1=0.
      b0=0.
      b0pr=0.
      bp2=-tf/4.
      bigp=-bp2/((1.-xk)*em4a-(xk*yo+1.))
      am1pr=2.*f*xk*(1.-xk)*bigp*em2a
      ap1pr=-am1pr
      am1=-2.*(1.-xk)*bigp*(e2a-em2a)
      bp2pr=xk*(yo+1.)*bigp
      bm2pr=-bp2pr
      bm2=bigp*((1.-xk)*e4a-xk*yo-1.)
      return
      end

```

EXAMPLE AND CHECK CASE: ELLIPSE

Stress Concentration around Elliptical Inclusion.

Program by Dave Campbell, US Geol. Survey, Denver. Feb77
Equations from donnell, 1941, von karman volume, pps.293-309.

Variables in namelist input:

xk = (Rigidity of inclusion)/(Rigidity of plate)
pri = Poisson's ratio of inclusion.
prp = Poisson's ratio of plate.
p1 = maximum principal stress on plate at infinity. (default=0.)
p3 = minimum principal stress on plate at infinity. (default=-1.)
angle = angle in degrees between p1 and ellipse major axis. (default=90)
(negative stresses are compressional, positive ones tensional.
hence default case has compression along ellipse's major axis.)
e = ellipticity of ellipse (=semi-minor axis/semi-major axis).
(Must be between 0.(thin sawcut) and 1.(circle). default=.5)
problem = 1 for plane strain (default),
2 for plane stress,
0 to stop.
detail = 0 to output stress concentration factor only. (default)
1 to output stress trajectory angles, too.
2 to output stress fields, too.
switch = 1 to normalize by mohr circle radius (default)
2 to normalize by hydrostatic stress (p1+p3)/2.

USER PROMPT MESSAGE

type namelist input:xk,pri,prp...(problem=0 to stop.)

INPUT RESPONSE

\$input xk=.1,prp=.25,pri=.4,detail=1\$

DIFFERENTIAL STRESS CONCENTRATION FACTORS AROUND AN ELLIPTICAL INCLUSION

plane strain calculation
inclusion rigidity/plate rigidity = 0.100
Poisson's ratio of plate = 0.250
Poisson's ratio of inclusion = 0.400
ellipticity of ellipse = 0.500
principal stress p1 = 0.000
principal stress p3 = -1.000
ellipse tilt angle = 90. deg

OUTPUT

theta=	0.00	10.00	20.00	30.00	40.00	50.00	60.00	70.00	80.00	90.00
alpha1	0.188	everywhere.....								
0.5491	-1.156	-0.841	0.228	0.550	1.019	1.341	1.551	1.681	1.751	1.773
0.6861	-0.528	-0.634	0.763	0.860	0.987	1.135	1.267	1.364	1.423	1.442
0.8221	0.441	0.667	0.939	1.064	1.102	1.123	1.152	1.185	1.210	1.219
0.9591	0.497	0.697	0.977	1.131	1.168	1.148	1.116	1.093	1.081	1.078
1.0951	0.585	0.733	0.977	1.135	1.185	1.160	1.105	1.049	1.011	0.997
1.2321	0.669	0.775	0.971	1.117	1.173	1.155	1.096	1.027	0.975	0.956
1.3681	0.741	0.816	0.968	1.093	1.150	1.139	1.085	1.016	0.961	0.940
1.5051	0.800	0.853	0.968	1.072	1.124	1.119	1.073	1.010	0.957	0.937
1.6411	0.846	0.884	0.971	1.054	1.100	1.099	1.061	1.006	0.960	0.941
1.7781	0.882	0.910	0.975	1.041	1.079	1.080	1.049	1.004	0.964	0.949
1.9141	0.909	0.930	0.979	1.031	1.062	1.064	1.040	1.003	0.970	0.957

stress trajectory: angle in degrees from x direction (along major axis)

theta=	0.00	10.00	20.00	30.00	40.00	50.00	60.00	70.00	80.00	90.00
alpha1	-90.00	125.40	100.25	58.98	63.87	70.11	75.75	80.81	85.50	90.00
0.5491	-90.00	81.30	79.63	78.55	77.94	78.89	81.05	83.83	86.87	90.00
0.6861	-90.00	76.04	77.76	81.45	83.84	85.08	86.00	87.10	88.47	90.00
0.8221	-90.00	77.39	78.33	82.39	85.94	88.17	89.28	89.72	89.89	90.00
0.9591	-90.00	80.10	79.83	83.20	86.82	89.53	91.01	91.36	90.89	90.00
1.0951	-90.00	82.59	81.60	84.09	87.34	90.08	91.74	92.15	91.42	90.00
1.2321	-90.00	84.54	83.29	85.02	87.74	90.27	91.93	92.38	91.59	90.00
1.3681	-90.00	85.98	84.74	85.90	88.10	90.30	91.83	92.27	91.54	90.00
1.5051	-90.00	87.03	85.93	86.69	88.43	90.28	91.61	92.02	91.37	90.00
1.6411	-90.00	87.79	86.87	87.37	88.72	90.24	91.36	91.71	91.17	90.00
1.7781	-90.00	88.35	87.60	87.93	88.97	90.19	91.12	91.41	90.96	90.00

USER PROMPT

grid points may now be changed via namelist grid.

variables in namelist grid:

theta = a 10-length vector of values theta as above.
alpha = an 11-length vector of values alpha as above.

INPUT RESPONSE

input namelist grid. (type "\$grid\$" for no change.)
\$grid\$

SECOND PROMPT

type namelist input:xk,pri,prp...(problem=0 to stop.)

RESPONSE TO END

\$input problem=0\$

PROGRAM

STOP

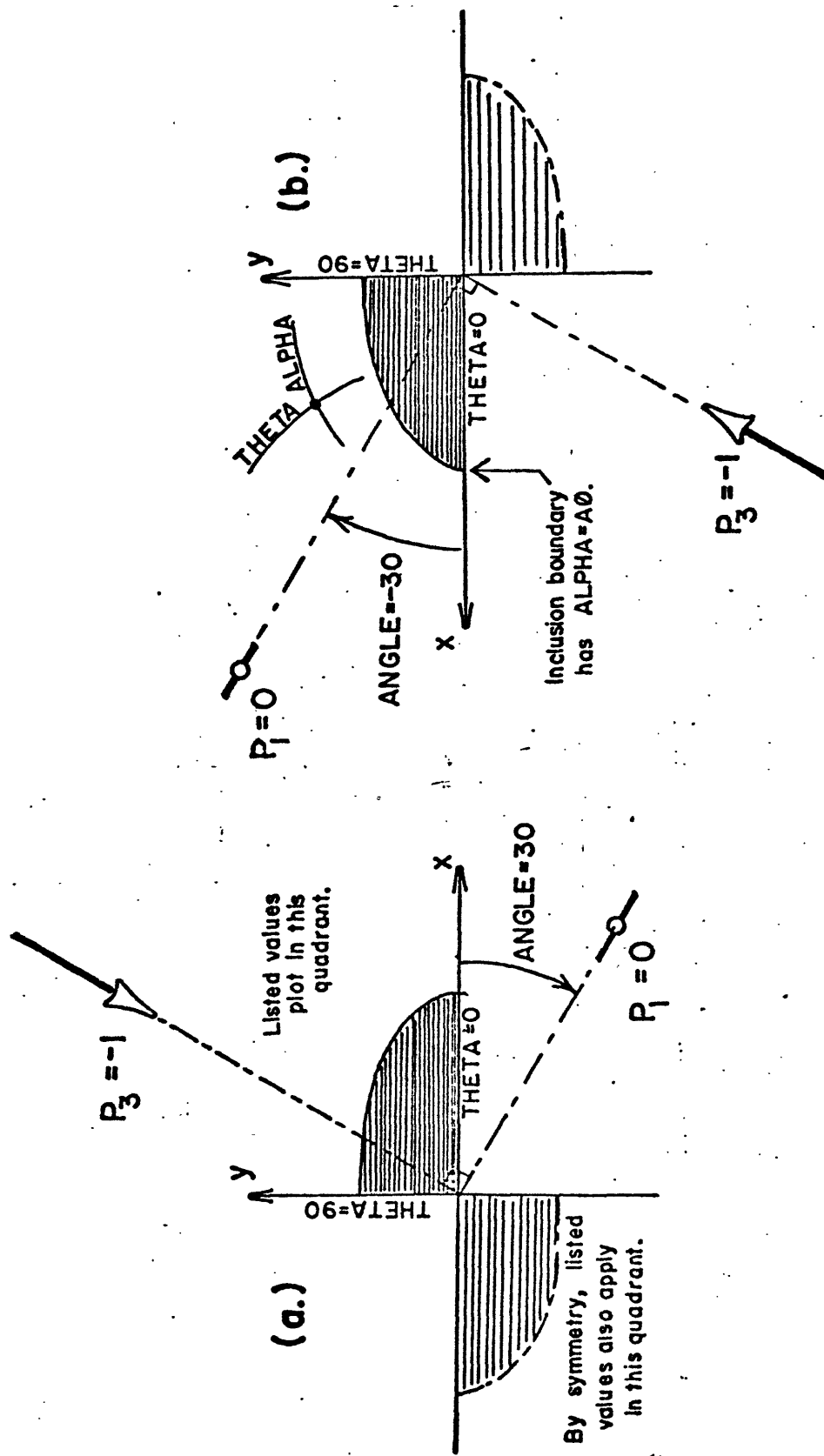


Figure 1 -- Geometry of ellipse axes (x, y) and directions of principal stress P_1 and P_3 . The case shown has $P_1 = 0$ and $P_3 = -1$ (uniaxial unit compression). ANGLE is the clockwise angle in degrees from the x-axis (ellipse major axis) to P_1 direction. (a) The case ANGLE = 30. (b) The case ANGLE = -30. In case (b) the direction of x has been reversed in order to show that both cases must be run to produce complete results which will cover the entire plane.